2023年度 大学院入学試験問題

2023 School Year Graduate School Entrance Examination Problem Booklet

システム情報学 / Information Physics and Computing

第 1 問から第 4 問のうち、 2 問のみを選択して解答せよ、 Answer two out of Problems 1-4.

試験時間 / Examination Time:

10:00~11:40

注 意 事 項 / Instructions

(1) 試験開始の合図まで、この問題冊子を開かないこと. Do not open this booklet until the starting signal is given.

the top of each answer sheet. Do never put your name.

(2) 本冊子に落丁、乱丁、印刷不鮮明の箇所などがあった場合には申し出ること、草稿用紙を含む全ページを本冊子から切り離してはならない.
You should notify the examiner if there are missing or incorrect pages in your booklet. Do not

separate all pages including the draft papers from this booklet.

- (3) 問題は第 1 問から第 4 問まであり、日本文は 2 頁から 18 頁、英文は 19 頁から 33 頁である。4 問のうち 2 問を日本語ないし英語で解答すること。 Four problems appear on pages 2 – 18 in Japanese and pages 19 - 33 in English. Answer two out of the four problems in Japanese or English.
- (4) 解答用紙 2 枚が渡される. 1 問ごとに必ず 1 枚の解答用紙を使用すること. 必要なときは解答用紙の裏面も使用してよい.
 Two answer sheets will be given Use one sheet per problem. You may use the back of the sheet
 - Two answer sheets will be given. Use one sheet per problem. You may use the back of the sheet if necessary.
- (5) 各解答用紙の指定された箇所に、受験番号およびその用紙で解答する問題番号を忘れずに記入すること、氏名は書いてはならない.
 Do not forget to fill the examinee's number and the problem number in the designated place at
- (6) 試験問題の内容に関する質問に対しては,原則として答えない. No questions relating to the contents of the problems are acceptable in principle.
- (7) 解答に関係のない記号, 符号などを記入した解答は無効とする.
 Any answer sheet with marks or symbols unrelated to the answer will be invalid.
- (8) 他に指示がない限り、問題解答用紙には最終解答のみでなく、計算や導出の過程を記述すること、論拠の不十分な解答は減点する、問題の解釈に複数の可能性が考えられる場合は、適宜言葉の定義や条件等を付加して解答してよい、

Your answers must include calculations and derivation processes, not just the final conclusion, unless otherwise indicated. Lack of justification will result in score deduction. In the case that a problem can be interpreted in several ways, you may answer the problem adding suitable definitions or conditions.

(9) 解答用紙および問題冊子は試験室から持ち出さないこと.
Do not take the answer sheets and this booklet out of the examination room.

受験番号		選択した問題番号
/ Examinee's	No.	/ Problem numbers
number		you selected

上欄に受験番号を記入すること.

Fill this box with your examinee's number

上欄に選択した2つの問題番号を記入すること。

Fill these boxes with the problem numbers you selected.

第1問

連続時間信号 $x_c(t)$ (t は時間を表す実数) をサンプリング周期T で標本化し、離散時間信号 x[n] (n は整数) を得た、すなわち、これらは $x[n]=x_c(nT)$ として関係付けられる、以下のように、 $x_c(t)$ のフーリエ変換 F_c を $X_c(\omega)$ 、x[n] の離散時間フーリエ変換 F_d を $X(\Omega)$ とする.

$$X_{c}(\omega) = \mathcal{F}_{c}[x_{c}(t)] = \int_{-\infty}^{\infty} x_{c}(t) e^{-j\omega t} dt$$

$$X(\Omega) = \mathcal{F}_{d}[x[n]] = \sum_{n=-\infty}^{\infty} x[n] e^{-j\Omega n}$$
(I)

ただし、 ω は角周波数、 $\Omega = \omega T$ は規格化角周波数である. 以下の問いに答えよ.

(1) $X_c(\omega)$ が以下の式で与えられるとする.

$$X_{c}(\omega) = \begin{cases} 1 - \frac{|\omega|}{B}, & |\omega| \leq B\\ 0, & \text{otherwise} \end{cases}$$
 (II)

ただし、Bは0より大きい実数である.このとき、x[n]が標本化によるエイリアシングを生じないためのTの条件を答えよ.

- (2) 式(I)の $X_c(\omega)$ と $X(\Omega)$ に関して、以下の問いに答えよ.
 - (a) $X_c(\omega)$ と $X(\Omega)$ について、以下の関係が成り立つことを示せ、

$$X(\Omega) = \frac{1}{T} \sum_{n=-\infty}^{\infty} X_{c} \left(\frac{\Omega}{T} - \frac{2\pi}{T} n \right)$$
 (III)

ただし、ディラックのデルタ関数 $\delta(t)$ に関する以下の関係式を用いてよい.

$$\int_{-\infty}^{\infty} \left\{ \sum_{n=-\infty}^{\infty} \delta(t - nT) \right\} e^{-j\omega t} dt = \frac{2\pi}{T} \sum_{n=-\infty}^{\infty} \delta\left(\omega - \frac{2\pi}{T}n\right)$$

(b) 式 (III) の関係を用いて、問い (1) のT の条件が与えられる理由を説明せよ、

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次に, x[n] の信号を M サンプル (M は 1 以上の整数) ごとに間引くことによって, 新たな離散時間信号 $x_{\rm d}[m]=x[Mm]$ (m は整数) を得た.

- (3) (a) $X_{c}(\omega)$ が式 (II) で与えられるとき、 $x_{d}[m]$ がエイリアシングを生じないためのTの条件を答えよ.
 - (b) $x_d[m]$ の離散時間フーリエ変換 $X_d(\Omega_d)$ ($\Omega_d = \omega MT$) を,元の信号 x[n] の離散時間フーリエ変換 $X(\Omega)$ を用いて表せ.
- (4) $X_{\rm c}(\omega)$ が式 (II) で与えられ, $B=\frac{\pi}{2T}$ であったとする.M=4 のとき, $x_{\rm d}[m]$ を求めよ.

第2問

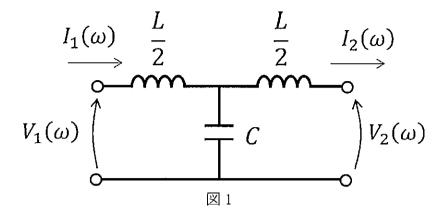
図 1 に示すようにインダクタンス L を二分する点にキャパシタンス C が接続された四端子回路を考える. 以下,自然対数の底を e,虚数単位を f,角周波数 e ω ,時間を f とし, $e^{j\omega t}$ に比例して正弦振動する電圧・電流の複素振幅 $V(\omega)$, $I(\omega)$ について考える.

(1) 入力端子側の電圧を $V_1(\omega)$, 電流を $I_1(\omega)$ とし、出力端子側の電圧を $V_2(\omega)$, 電流を $I_2(\omega)$ と表す. このとき、入力側と出力側を関係づける次の行列要素 a,b,c,d を、 ω,L,C を用いて表せ.

$$\begin{bmatrix} V_1(\omega) \\ I_1(\omega) \end{bmatrix} = \begin{bmatrix} a & b \\ c & d \end{bmatrix} \begin{bmatrix} V_2(\omega) \\ I_2(\omega) \end{bmatrix}$$

- (2) 出力端子を開放する(つまり $V_2(\omega)/I_2(\omega)$ を無限大とする)とき、以下の問いに答えよ.
 - (a) 周波数伝達関数 $G_o(\omega) = V_2(\omega)/V_1(\omega)$ を, ω, L, C を用いて表せ.
 - (b) 問い(2)-(a)で求めた周波数伝達関数のゲイン $|G_o(\omega)|$ について, $\omega \geq 0$ の範囲で概形を描け.
- (3) 出力端子をある周波数特性を有するインピーダンス $Z(\omega)$ で終端したところ,入力インピーダンス $V_1(\omega)/I_1(\omega)$ が $Z(\omega)$ に等しくなったとする.このような $Z(\omega)$ を, ω,L,C を用いて表せ.ただし, $Z(\omega)$ が根号を有する場合,正符号のみを回答すればよい.(なお,この $Z(\omega)$ は,図 1 の回路を無数に縦続接続してできる梯子状回路の入力インピーダンスを表す.)
- (4) 出力端子を問い(3)で求めた $Z(\omega)$ で終端するとき、以下の問いに答えよ、
 - (a) 周波数伝達関数のゲイン $|G_L(\omega)| = |V_2(\omega)/V_1(\omega)|$ を, $0 \le \omega \le \sqrt{4/LC}$ と $\omega > \sqrt{4/LC}$ とで場合分けして求め, ω, L, C を用いて表せ.
 - (b) 問い(4)-(a)で求めた周波数伝達関数のゲイン $|G_L(\omega)|$ について, $\omega \geq 0$ の範囲で概形を描け.

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第3問

図 1 に示すように、制御対象 P(s)、制御器 C(s) からなる閉ループ系を考える. $P(s) = \frac{s-1}{(s+6)(s-3)}$ とし、以下の問いに答えよ.

- (1) $C(s) = C_1(s) = \frac{K}{s-2}$ とする. ただし K > 0 とする.
 - (a) 閉ループ系の特性方程式を $a_1(s) = a_{11}(s) + Ka_{12}(s) = 0$ とする. ただし $a_1(s), a_{11}(s), a_{12}(s)$ はs の多項式である. $a_{11}(s), a_{12}(s)$ を示せ.
 - (b) 閉ループ系が安定となる K の条件を示せ.
- (2) $C(s) = C_2(s) = \frac{K}{s+2}$ とする. ただし K > 0 とする.
 - (a) 閉ループ系の特性方程式を $a_2(s)=a_{21}(s)+Ka_{22}(s)=0$ とする. ただし $a_2(s), a_{21}(s), a_{22}(s)$ は s の多項式である. $a_{21}(s), a_{22}(s)$ を示せ.
 - (b) $a_2(1)$, $a_2(+\infty)$ のそれぞれの符号を示せ. またその結果から、閉ループ系の極について、その安定性の観点から説明せよ.
- (3) ある安定でプロパーな伝達関数 $G(s)=\frac{n(s)}{d(s)},\ n(s)$: ℓ 次多項式, d(s): m 次多項式, $\ell \leq m$, を考える. G(s) が零点 s=z を持ち, それは実数で z>0, かつ $C_+:=\{s\,|\, \mathrm{Re}[s]\geq 0\}$ の領域では無限遠点の零点を除いて唯一とする.
 - (a) G(s) が $G(s)=G_o(s)\frac{s-z}{s+z}$ と表せることを説明せよ.ただし $G_o(s)$ は安定でプロパー,無限遠点の零点を除く $G_o(s)$ の全ての零点 $\{z_i\}$ は $\mathrm{Re}[z_i]<0$ を満す.
 - (b) 次が成り立つことを示せ.

$$|G(j\omega)| = |G_o(j\omega)|, \ \forall \omega \in (-\infty, +\infty)$$

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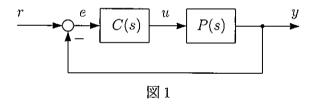
(4) C(s) を問い(1)の $C(s) = C_1(s)$ とし、閉ループ系が安定な場合を考える.その相補感度関数を次に与える.

$$T(s) = \frac{P(s)C_1(s)}{1 + P(s)C_1(s)}$$

- (a) T(2) の値を示せ.
- (b) 問い(3)の結果を参考にして次が成り立つことを示せ.

$$\sup_{\omega \in (-\infty, +\infty)} |T(j\omega)| > 3$$

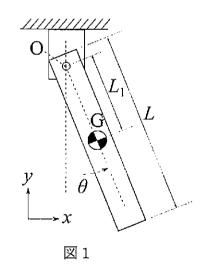
(c) 制御系設計の観点から、問い (4)-(b) の結果を P(s) や C(s) の零点や極とあわせて説明せよ.

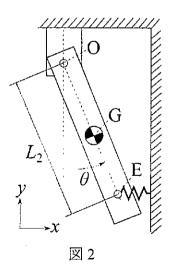


第4問

図 1 に示す一様密度の剛体棒の運動を考える. 剛体棒は質量 M [kg], 長さ L [m], 重心を G とする. 剛体棒は静止節(天井)と回転軸 O で接続されており、O を中心として xy 平面上で振動する. 棒の太さと、O 周りの回転摩擦は十分小さく無視できるものとする. OG 間の距離は L_1 ($0 < L_1 < L/2$), 重力加速度を g [m/s²]とする. 図 1 中,直交座標系 xy および剛体棒の回転角度 θ [rad]を定義しており,それぞれ矢印の向きを正とする. y 軸負の方向が重力方向である. これらの条件下で,以下の問いに答えよ.

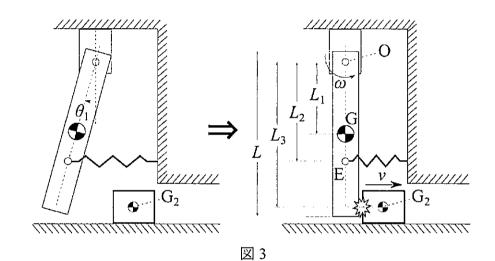
- (1) G 周りの剛体棒の慣性モーメント I を, M, L を用いて表せ、また、O 周りの剛体棒の慣性モーメント I' を, M, L, L_1 を用いて表せ、 (問い (2) 以後では、式の簡略化のために I と I'は用いてよい。)
- (2) (a) 図1について、O周りの剛体棒の回転の運動方程式を導出せよ.
 - (b) 図 2 のように剛体棒上の点 E と静止節(垂直壁面)の間に ばね定数 k [N/m]の線形ばねを x 軸と水平に取り付けた。OE 間の距離は L_2 ($0 \le L_2 \le L/2 + L_1$) とする. ばねは剛体棒が静止 ($\theta = 0$) しているときに自然長とし、剛体棒が振動するときに x 軸方向の変形が生じる. ばねの y 軸方向の変形は十分小さく無視できるものとする. これらの条件下で、O周りの剛体棒の回転の運動方程式を導出せよ. また、振動の角度 θ が微小であるとき、剛体棒の固有振動数を求めよ.





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- (3) 問い (2) で使用したばね付きの剛体棒を図 3 のように傾けたのち開放し、初期角度 $\theta = \theta_1$ (< 0)、初期角速度 $\dot{\theta} = 0$ で振動を開始する. θ_1 は微小とする. その後、剛体棒が $\theta = 0$ 、 $\dot{\theta} = \omega$ (> 0) のときに、質点とみなせる小物体と完全弾性衝突(はね返り係数 e = 1)する場合を考える. 小物体の質量を m [kg]、中心を G_2 とし、x 軸と水平な床から見て衝突の接触点は G_2 と同じ高さとし、衝突接触点と G_2 の距離は G_3 と同じ高さとし、衝突接触点と G_3 の距離は G_3 の多件下で、衝突直前の剛体棒の角速度 G_3 と、衝突直後の小物体の並進速度 G_3 (> 0) G_3 (G_3) G_4 (G_5) G_5 (G_5) G_5 (G_5) G_5 (G_5) G_5) G_5 (G_5) G_5) G_5 (G_5) G_5) G_5 G_5 G_5) G_5 $G_$
- (4) 衝突後, 小物体は摩擦のある床面上を滑る. これまでに導出した式と, 長さの条件を考慮して, 小物体の移動距離を最大化する長さ L_2 と L_3 を求めよ. ただし, L と L_1 は定数とし, M=m とする.



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2023 IPC

Problem 1

A discrete-time signal x[n] (n is an integer number) is obtained by sampling a continuous-time signal $x_{\rm c}(t)$ (t is a real number representing time) with the sampling period T. Therefore, these signals are related as $x[n] = x_{\rm c}(nT)$. The Fourier transform $\mathcal{F}_{\rm c}$ of $x_{\rm c}(t)$ and discrete-time Fourier transform $\mathcal{F}_{\rm d}$ of x[n] are denoted as $X_{\rm c}(\omega)$ and $X(\Omega)$, respectively, as follows:

$$X_{c}(\omega) = \mathcal{F}_{c}[x_{c}(t)] = \int_{-\infty}^{\infty} x_{c}(t) e^{-j\omega t} dt$$

$$X(\Omega) = \mathcal{F}_{d}[x[n]] = \sum_{n=-\infty}^{\infty} x[n] e^{-j\Omega n}$$
(I)

Here, ω is the angular frequency, and $\Omega = \omega T$ is the normalized angular frequency. Answer the following questions.

(1) Suppose that $X_{c}(\omega)$ is given by

$$X_{c}(\omega) = \begin{cases} 1 - \frac{|\omega|}{B}, & |\omega| \le B \\ 0, & \text{otherwise} \end{cases}, \tag{II}$$

where B is a real number larger than 0. Answer the condition for T such that aliasing due to sampling does not occur in x[n].

- (2) Answer the following questions about $X_c(\omega)$ and $X(\Omega)$ in Equation (I).
 - (a) Prove that the following relationship between $X_c(\omega)$ and $X(\Omega)$ holds:

$$X(\Omega) = \frac{1}{T} \sum_{n=-\infty}^{\infty} X_{c} \left(\frac{\Omega}{T} - \frac{2\pi}{T} n \right)$$
 (III)

Here, the following equation including the Dirac delta function $\delta(t)$ can be used:

$$\int_{-\infty}^{\infty} \left\{ \sum_{n=-\infty}^{\infty} \delta(t - nT) \right\} e^{-j\omega t} dt = \frac{2\pi}{T} \sum_{n=-\infty}^{\infty} \delta\left(\omega - \frac{2\pi}{T}n\right)$$

(b) Using Equation (\mathbb{H}), justify the condition for T obtained in Question (1).

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Next, by decimating x[n] by a factor of M (M is an integer number equal to or larger than 1), a new discrete-time signal $x_{\rm d}[m] = x[Mm]$ (m is an integer number) is obtained.

- (3) (a) Answer the condition for T such that aliasing does not occur in $x_{\rm d}[m]$ when $X_{\rm c}(\omega)$ is given by Equation (II).
 - (b) Express the discrete-time Fourier transform of $x_d[m]$, $X_d(\Omega_d)$ ($\Omega_d = \omega MT$), in terms of the discrete-time Fourier transform of the original discrete-time signal x[n], $X(\Omega)$.
- (4) Suppose that $X_{\rm c}(\omega)$ is given by Equation (II) and $B = \frac{\pi}{2T}$. Obtain $x_{\rm d}[m]$ for M = 4.

Problem 2

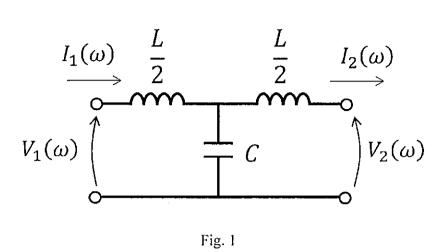
Consider a four-terminal circuit composed of capacitance C connected to the midpoint of inductance L as shown in Fig. 1. The response of the circuit is described in terms of a set of complex voltages and currents, with complex amplitudes $V(\omega)$ and $I(\omega)$, respectively, oscillating in proportion to $e^{j\omega t}$, where e is the base of natural logarithms, e is the imaginary unit, e is an angular frequency, and e is time.

(1) We express the voltage and current at the input port as $V_1(\omega)$ and $I_1(\omega)$ and those at the output port as $V_2(\omega)$ and $I_2(\omega)$. Express elements a, b, c, d of the matrix in the equation below in terms of ω, L, C .

$$\begin{bmatrix} V_1(\omega) \\ I_1(\omega) \end{bmatrix} = \begin{bmatrix} a & b \\ c & d \end{bmatrix} \begin{bmatrix} V_2(\omega) \\ I_2(\omega) \end{bmatrix}$$

- (2) Answer the following questions assuming the output port is open, i.e. $V_2(\omega)/I_2(\omega)$ is set to be infinite.
 - (a) Express the frequency transfer function $G_0(\omega) = V_2(\omega)/V_1(\omega)$ using ω, L, C .
 - (b) Draw the outline of the gain of the frequency transfer function obtained in Question (2)-(a), $|G_o(\omega)|$, for $\omega \ge 0$.
- (3) Suppose that the input impedance $V_1(\omega)/I_1(\omega)$ is equal to $Z(\omega)$ when the output port is terminated with an impedance with a given frequency characteristic $Z(\omega)$. Express such a $Z(\omega)$ using ω, L, C . If a square root is used in the expression, consider only the positive root. (Note that this $Z(\omega)$ expresses the input impedance of a ladder network composed of an infinite cascade of the circuit shown in Fig. 1.)
- (4) Answer the following questions assuming the output port is terminated with the impedance function $Z(\omega)$ obtained in Question (3).
 - (a) Express the gain of the frequency transfer function $|G_L(\omega)| = |V_2(\omega)/V_1(\omega)|$ using ω, L, C . The answer should be expressed separately for cases of $0 \le \omega \le \sqrt{4/LC}$ and $\omega > \sqrt{4/LC}$.
 - (b) Draw the outline of the gain of the frequency transfer function $|G_L(\omega)|$ obtained in Question (4)-(a) for $\omega \geq 0$.

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Problem 3

Consider the feedback control system composed of plant P(s) and controller C(s) as shown in Fig. 1. Suppose $P(s) = \frac{s-1}{(s+6)(s-3)}$ and answer the following questions.

- (1) Suppose that $C(s) = C_1(s) = \frac{K}{s-2}$, where K > 0.
 - (a) Let $a_1(s) = a_{11}(s) + Ka_{12}(s) = 0$ be the characteristic equation of the closed loop system, where $a_1(s)$, $a_{11}(s)$, and $a_{12}(s)$ are polynomials of s. Find expressions for $a_{11}(s)$ and $a_{12}(s)$.
 - (b) Determine the condition(s) on K such that the closed loop system is stable.
- (2) Suppose that $C(s) = C_2(s) = \frac{K}{s+2}$, where K > 0.
 - (a) Let $a_2(s) = a_{21}(s) + Ka_{22}(s) = 0$ be the characteristic equation of the closed loop system, where $a_2(s)$, $a_{21}(s)$, and $a_{22}(s)$ are polynomials of s. Find expressions for $a_{21}(s)$ and $a_{22}(s)$.
 - (b) Determine the signs of $a_2(1)$ and $a_2(+\infty)$. Given the results, explain the poles of the closed loop system with respect to its stability.
- (3) Consider a stable and proper transfer function $G(s) = \frac{n(s)}{d(s)}$, where n(s) and d(s) are polynomials of degree ℓ and m, respectively, and $\ell \leq m$. Suppose that G(s) has a zero s = z, where z is real, z > 0, and it is unique in the region $C_+ := \{s \mid \text{Re}[s] \geq 0\}$ except for zeros at infinity.
 - (a) Explain how G(s) can be represented as $G(s) = G_o(s) \frac{s-z}{s+z}$, where $G_o(s)$ is stable and proper, and all the zeros $\{z_i\}$ of $G_o(s)$ except for zeros at infinity satisfy $\text{Re}[z_i] < 0$.
 - (b) Prove the following:

$$|G(j\omega)| = |G_o(j\omega)|, \ \forall \omega \in (-\infty, +\infty)$$

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(4) Suppose that $C(s) = C_1(s)$ in Question (1) and that the closed loop system is stable. Its complementary sensitivity function is given by

$$T(s) = \frac{P(s)C_1(s)}{1 + P(s)C_1(s)}.$$

- (a) Show the value of T(2).
- (b) Prove the following based on the results of Question (3):

$$\sup_{\omega \in (-\infty, +\infty)} |T(j\omega)| > 3$$

(c) Explain the result of Question (4)-(b) with the zeros and/or the poles of P(s) and/or C(s) with respect to control system design.

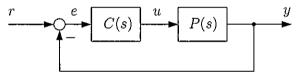


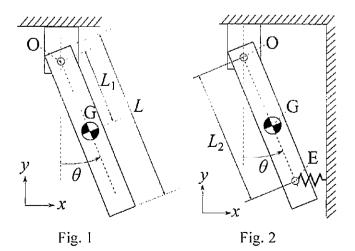
Fig. 1

Problem 4

Consider the motion of a rigid rod with uniform density as shown in Fig. 1. The rigid rod has mass M [kg], length L [m], and center of gravity G. The rigid rod connects to a fixed base link (the ceiling) by the rotary joint O and oscillates on the xy plane about O. The thickness of the rod and the rotational friction around O are sufficiently small and can be considered negligible. The distance between O and G is L_1 ($0 < L_1 < L/2$), and the gravitational acceleration is g [m/s²]. Note that the Cartesian coordinate system xy and the rotation angle θ [rad] of the rigid rod are defined in Fig. 1. The directions of the arrows indicate positive directions for the aforementioned quantities. The negative direction of y-axis is the direction of gravity. Under these conditions, answer the following questions.

- (1) Express the moment of inertia I of the rigid rod around G using M and L. Also, express the moment of inertia I' of the rigid rod around O using M, L, and L_1 . (From Question (2) onward, I and I' may be used to simplify equations.)
- (2) (a) For Fig. 1, derive the equation of motion of the rotation of the rigid rod about O.
 - (b) As shown in Fig. 2, a linear spring with spring constant k [N/m] was attached horizontally, along the x-axis, between the point E on the rigid rod and a fixed base link (the vertical wall surface). The distance between O and E is L_2 ($0 \le L_2 \le L/2 + L_1$). The spring has a natural length when the rigid rod is stationary ($\theta = 0$), and deformation occurs in the x-axis direction when the rigid rod oscillates. The deformation of the spring in the y-axis direction is sufficiently small and can be considered negligible. Under these conditions, derive the

equation of motion of the rotation of the rigid rod about O. Also, derive the natural frequency of the rigid rod when the angle θ of oscillation is very small.



(Continued on next page)

(3) After tilting and releasing the rigid rod with the spring used in Question (2) as shown in Fig. 3, the rod starts oscillating from an initial angle θ = θ₁ (< 0) and initial angular velocity θ = 0. θ₁ is assumed as very small. After that, consider the case where the rigid rod collides in a perfectly elastic collision (with a rebound coefficient of e = 1) with a small object assumed as a point mass when θ = 0 and θ = ω (> 0). The mass of the small object is m [kg], and its center is G₂. The contact point of the collision is at the same height as G₂ from the floor (which is parallel to the x-axis), and the distance between the collision contact point and O is L₃ (0 ≤ L₃ ≤ L/2 + L₁).

Under these conditions, derive the angular velocity ω of the rigid rod immediately before the collision and the translational velocity ν (> 0) [m/s] of the small object immediately after the collision.

(4) After the collision, the small object slides on the floor surface with friction. Determine the lengths L_2 and L_3 that maximize the moving distance of the small object considering the equations derived in the previous questions and the length conditions. L and L_1 are assumed as constants, and M=m.

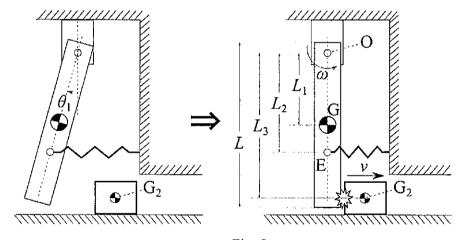


Fig. 3